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IN THE SPECIFICATION:

Please enter the amendments to the Specification of the above-

identified application as follows.

Please amend the paragraph beginning on page 3, line 21 as follows.

--Another operational drawback of conventional combustion

powered combustion-powered tools, is that when operated at relatively higher

elevations or altitudes, there is less air for combustion. As a result, when used at

such higher elevations, conventional combustion-powered tools with constant

volume fuel metering valves can have overly rich fuel/air mixtures in their

combustion chambers, which can lead to fouling of the ignition system as well as

other operational difficulties. As such, there is a need for a combustion

powered combustion-powered tool with a fuel metering valve which has the

capability of adjusting the amount of fuel in the combustible fuel/air mixture.--

Please amend the paragraph beginning on page 5, line 5 as follows.

--More specifically, the present invention provides a variable volume

metering chamber and valve assembly for a combustion-powered tool

includes including a housing defining a metering chamber having an internal

volume and including an inlet and an outlet, and a plunger configured for

reciprocal movement relative to the chamber for adjusting the internal volume of

the metering chamber. The plunger is preferably adjustable by the user to alter the

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volume of fuel retained in the metering chamber. In the housing, a first valve controls control fluid flow through the inlet, a second valve controls fluid flow through the outlet, and an actuator assembly, connected to the valves, is sequentially operable from a first position, in which the first valve is open and the second valve is closed, to a second position, in which the first and second valve are both closed, and a third position, in which the first valve is closed and the second valve is open.--152 40

Please amend the paragraph beginning on page 10, line 12 as follows.

The spring-biased valves, 16, 18 each include a preferably conical seat 30, 32, a rod 34, 36, and a spring 38, 40, respectively. Although discussed in terms of the first spring-biased valve 16, it is to be understood that the following description also applies to the corresponding parts of the second spring-biased valve 18. The seat 30 is sized and configured to sealingly engage with the inlet 20 of the metering chamber 24 when the spring-biased valve 16 is in a closed position. Movement of the seat 30 between an open position and the closed position, is controlled by the rod 34. Although the spring 38 is an economical method of biasing the valve, use of other biasing devices is contemplated. The spring 38 is used to bias the valve 16 toward the closed position. Each of the springs 38, 40 has an anchored end 42, 44 and a movable end 46, 48, respectively. The movable end 46 exerts a force against the seat 30 tending to move it in the

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direction of the metering chamber 24 by the force of the spring 4038 pushing against the anchored end 42. Although the anchored end 42 may be anchored directly to the housing 12, preferably, the anchored end is seated within a compartment described in greater detail below.--

Please amend the paragraph beginning on page 17, line 9 as follows.

--In the preferred embodiment, this operation or valve sequence is controlled by the pivoting action of the link arm 68 which moves the actuator assembly 60 from a position where the upper arm 62 has a maximum spacing from the housing 12 (FIG. 3A), to a position where the lower arm 64 has a maximum spacing from the housing 12 (FIG. 3C). In the preferred embodiment, in addition to the loose mating engagement between the notch 67 and the tongue 69; the actuator assembly 60 also includes a delay mechanism also operating between the closing of one of the valves 16, 18 and the closing of the other valve 18, 16. Any type of delay mechanism is suitable, such as an electrical delay, electronic means ofor a mechanical delay mechanism. In the most preferred mechanical delay mechanism, the actuator assembly 60 is slidably connected to each of the rods 34, 36. The first rod 34 has a first opener 71 such as a 'C'-clip secured to the rod-34 and the second rod 36 has a second opener 72. Spacing of the openers 71, 72 on the rods 34, 36 are preferably used to create a delay in the closing of one valve 16, 18 before the opening of the other valve 18, 16.--

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Please amend the paragraph beginning on page 18, line 1 as follows.

--In the preferred delaying mechanism, the control arm 66 of the actuator assembly 60 is longer than the housing 2612 in which the valve assembly 10 resides. The excess length is sufficient to allow the upper arm 62 and the lower arm 64 to sandwich the housing 12 between them with excess space between the housing, and the actuator arms 62, 64. In response to the stimulus that triggers the valve sequences, the control arm 66 moves up and down (directions relate to the tool, as oriented in FIG. 3).--

Please amend the paragraph beginning on page 19, line 3 as follows.

-Seals are used where suitable to prevent flow of the fluid into the area outside the valve assembly 10, the metering chamber 24, and the housing 12. The exact number, shape and placement of such seals depend on the exact configuration of the valve assembly 10 for a specific application. In the preferred embodiment shown, a removable insert 74 is optionally used to surround the rod 34, 36 of each of the spring-biased valves 16, 18 as the rod passes through the housing 2612 and contacts actuator assembly 60. O-rings 76, gaskets or similar devices, are preferably used to prevent leakage between the removable insert 74 and the housing 12 or the rods 34, 36. In some applications, it will be preferable for the length of the spring 38, 40 to exceed the dimensions of the upper chamber 54 or the lower chamber 56. When this is desirable, the removable insert 74 includes a hollow compartment 78 that is sized and configured to receive a portion

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of the length of the spring 38, 40, and to receive the anchored end 42. The removable insert 74 also provides easy access to the spring-biased valves 16, 18 and their component parts when replacements are installed.--

Please amend the paragraph beginning on page 23, line 14 as follows.

--In use, the canister 28 is inserted into the combustion tool so that the nipple 51 matingly engages the adapter 96. The canister 28 is pressed upon the nipple 51 so that the membrane 100 is pierced and the nipple end 104 enters the passageway 98 until contact is made with the stem end 102. As described above, as a sealing relationship is preferably obtained, and it is contemplated that other locking apparatus may be employed to secure the canister 28 in this position.--

Please amend the paragraph beginning on page 23, line 20 as follows.

--Thus, it will be seen by those skilled in the art that the present valve assembly and metering ehangerchamber provide a simple method of providing a constant volume of fluid to a power fastening tool. The two spring-biased valves 16, 18 control the inlet and the outlet to the constant volume metering chamber 24, measuring a constant amount of fluid, independent of in fluctuations in the fluid flow rate. The actuator assembly 60 manipulates opening and closing of the valves 16, 18, receiving the fluid from the pressurized source 28

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and metering it before it flows downstream to a combustion or expansion

chamber. This arrangement of the valves 16, 18 minimizes wear on the seals,

reducing maintenance.--

Please amend the paragraph beginning on page 26, line 9 as follows.

-- So that the operation of the valve 16, 18 is not impaired, it is

preferred that the plunger 122 be disposed in the metering chamber 24 in an offset

position. In other words, the longitudinal axis of the plunger 122 is offset from a

vertical plane bisecting the metering chamber 24 in the direction of reciprocal

movement of the plunger. Practically speaking, and referring now to FIG. 6, the

plunger 122 is located behind the axis of movement of the valves 16, 18.--

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